The "5 Pillars" of Residential A/C Refrigerant Circuit Diagnosis

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| Fixed Metering | Suction Pressure | Head Pressure | Super heat | Subcool | Comp. Amps | Δt |
|---------------------------------|---------------------|------------------|---------------|---------|---------------|----------|
| Low Charge | Ŷ | Ŷ | Î | Ŷ | Ŷ | Ŷ |
| Over Charge | Ŷ | Î | Ŷ | Ŷ | Ŷ | Norm |
| Low Indoor Airflow/Low R/A Temp | Ŷ | Norm | Ŷ | Norm | Norm | î |
| Dirty Condenser | Ŷ | t | Norm | Norm | t | Ŷ |
| Liquid Line Restriction* | Ŷ | Norm | t | Norm | Ŷ | Ŷ |
| Oversized Piston | Norm | Norm | Ŷ | Ŷ | Ŷ | Ŷ |
| High R/A Temp | t | t | Î | Norm | Ŷ | Ŷ |

| TXV System | Suction Pressure | Head Pressure | Super heat | Subcool | Comp. Amps | Δt |
|---|---------------------|------------------|---------------|---------|---------------|------|
| Liquid Line Restriction* | Ŷ | Norm | Ŷ | Norm | Ŷ | Ŷ |
| Overfeed / Loose Bulb Bulb Insulated | 1 | Ŷ | Ŷ | Ŷ | Ŷ | Ŷ |
| Low Charge Slightly | Norm | Ŷ | Norm | Ŷ | Ŷ | Norm |
| Over Charge Slightly | Norm | 1 | Norm | Î | t | Norm |
| Low Indoor Airflow/Low R/A Temp | Ŷ | Norm | Ŷ | Norm | Norm | t |

*Restriction effect on head pressure varies with the exact location of the restriction, system refrigerant capacity, presence of a receiver and whether the charge is correct. Restrictions may show high head with short runtime, or on systems with very small condensers or on systems that have been overcharged in addition to the restriction. In general, restrictions are in the liquid Line and will result in low head after sufficient run time.

Suction pressure, head pressure, subcooling, superheat, delta T.

Taking all five of these calculations into account on every service call is critical. Even if you must do further diagnostic tests to pinpoint the problem, these five factors are the groundwork before more effective diagnostics can be done. I would also add static pressure as an important reading that should be checked regularly (keep TESP between 0.3"WC and 0.7"WC on most systems), but I would still place it slightly below these five as far as fundamental HVAC technician measurements.

Some of these are "rules of thumb" and are obviously for reference only. Refer to manufacturer recommendations when setting a charge.

Suction Pressure/Low Side

Suction pressure tells us several things. The first thing it tells us is what the boiling temperature of the refrigerant in the evaporator is. If the suction pressure is below 32° saturation temperature, the evaporator coil will eventually freeze.

As a general rule, the higher the temperature of the air passing over the evaporator, the higher your suction pressure will be. A good rule of thumb for suction pressure is 35° saturation below indoor ambient +/- 5° (return temperature measured at the evaporator coil). This temperature differential is often called an evaporator split or design temperature difference (DTD). When calculating DTD, a "higher" DTD means lower suction pressure compared to the return temperature; a lower DTD means higher suction pressure.

That means that when the temperature of the air passing over the evaporator is 80°, the low side saturation temperature should be 45° when the system is set for 400 CFM per ton output. Remember, the temperature scale next to the pressure scale on the gauge represents saturation. If you don't have the correct scale on your gauge (or in your gauge, if you have a <u>digital manifold</u>), you would need to use a PT chart.

The 35° rule only works at 400 CFM per ton. When a system is designed for 350 CFM per ton, the DTD will be closer to $38-40^{\circ}+/-5^{\circ}$.

Make sure you know the actual CFM output of the system before you calculate DTD. It can vary significantly based on the setup of the particular blower. Also, keep in mind that oversized evaporator coils that some manufacturers specify for efficiency can also result in slightly lower DTD (higher suction). If you don't know all the details, it is my experience that using 35° is the best bet.

Head Pressure/High Side

When used in conjunction with liquid line temperature, we can know the state of the refrigerant in the liquid line and that the compressor is pumping/operating in the required compression ratio. We can also know something about the state of the metering device—as to whether or not refrigerant is "backing up" against the metering device. A good rule of

thumb for head pressure is a 15-20° saturation above outdoor ambient +/- 3° for most modern systems. These saturation/ambient calculations are only indicators; they are not set in stone. Keep in mind that when I say "ambient," I am talking about the air entering the evaporator for suction pressure and the condenser for head pressure.

Jim Bergmann points out that different equipment efficiencies will have different target condensing temperature over ambient (CTOA) readings. Keep in mind that these date ranges don't guarantee the SEER but rather give the date ranges that these efficiencies will be most likely. The larger the condenser coil is in relation to the volume of refrigerant being moved, the lower the CTOA will be.

- 6 10 SEER Equipment (Older than 1991) = 30° CTOA
- 10 -12 SEER Equipment (1992 2005) = 25° CTOA
- 13 15 SEER Equipment (2006 Present) = 20° CTOA

16 SEER+ Equipment (2006 – Present) = 15° CTOA

Superheat

Superheat is important for two reasons. It tells us whether or not we could be damaging the compressor and whether we are fully feeding the evaporator with boiling, flashing refrigerant. If the system has a 0° superheat, a mixture of liquid and vapor is entering the compressor. This is called liquid slugging, and it can damage a compressor. A superheat that is higher than the manufacturer's specification can both starve the evaporator, causing capacity loss, as well as cause the compressor to overheat. So, how do we know what superheat we should have? First, we must find out what type of metering device the system is using. If it is using a piston or other fixed metering device, you must refer to the manufacturer's superheat requirements or a superheat chart like the one below.

Superheat for A/C with fixed Orifice R-22

| | | LIV | apor | ator | 1111 | ci A | | emp | ciat | ure | r ann | enn | en | ret | Duit | y (en | unaip |
|--|----|-----|------|------|------|------|----|-----|------|-----|-------|-----|----|-----|------|-------|-------|
| | 50 | 52 | 54 | 56 | 58 | 60 | 62 | 64 | 66 | 68 | 70 | 72 | 74 | 76 | 78 | 80 | 82 |
| Outside Air Temperature Dry bulb | | | | | | | | | | | | | | | | | |
| 60 | 7 | 11 | 13 | 17 | 18 | 20 | 24 | 26 | 28 | 30 | | | | Û. | | | |
| 65 | | 8 | 11 | 13 | 15 | 17 | 18 | 22 | 25 | 28 | 30 | | | | | | |
| 70 | | | 8 | 11 | 12 | 14 | 16 | 18 | 22 | 25 | 28 | 30 | | | | | |
| 75 | | | | 7 | 10 | 12 | 14 | 16 | 18 | 23 | 26 | 28 | 30 | | | | |
| 80 | 0 | | | | 6 | 8 | 12 | 14 | 16 | 18 | 23 | 27 | 28 | 30 | | | |
| 85 | | | | | | 6 | 8 | 12 | 14 | 17 | 20 | 25 | 27 | 28 | 30 | - | |
| 90 | | | | | | | 6 | 9 | 12 | 15 | 18 | 22 | 25 | 27 | 28 | 30 | |
| 95 | | | | | | | | 7 | 11 | 13 | 16 | 20 | 23 | 24 | 26 | 28 | 30 |
| 100 | | | | | | | | 5 | 8 | 11 | 14 | 18 | 20 | 22 | 24 | 26 | 28 |
| 105 | | | | | | | | | 6 | 8 | 12 | 15 | 19 | 21 | 23 | 25 | 26 |
| 110 | | | | | | | | | 5 | 7 | 11 | 12 | 17 | 19 | 21 | 23 | 25 |
| 115 | | | | | | | | | | 5 | 8 | 13 | 15 | 17 | 20 | 22 | 23 |

Evaporator Inlet Air Temperature Fahrenheit Wet Bulb (enthalpy)

If it is a TXV-type metering device, the TXV will generally attempt to maintain between a 5–15° superheat on the suction line exiting the evaporator coil (10° +/- 5°).

TXV target superheat setting may vary slightly based on equipment type.

Subcooling

Subcooling tells us whether the liquid line is full of liquid or not. A 0° subcool reading tells us that the refrigerant in the liquid line is part liquid and part vapor. An abnormally high subcool reading tells us that the refrigerant is moving through the condenser too slowly, causing it to give up a large amount of sensible heat past saturation temperature. A high subcool is often accompanied by high head pressure and, conversely, a low subcool by low head pressure. Subcool is always a very important calculation to take because it lets you know whether or not the metering device is receiving a full line of liquid. Typical ranges for subcooling are between 8 and 14 degrees on a TXV system, but always check the manufacturer's information to confirm. In general, on a TXV system using 10°, +/- 3° at the condenser outlet is an acceptable "rule of thumb" in the absence of manufacturer's data.

On a fixed orifice/piston system, the subcooling will vary even more based on load conditions, and you will see a range of $5-23^{\circ}$, making subcooling less valuable on a fixed orifice system. In my experience, during normal operating conditions, the subcooling on a fixed orifice system will still usually be in the 10° +/- 3° range.

Evaporator Air Temperature Split (Delta T)

The evaporator air temperature split (delta T) is a nice calculation because it gives you a good look at system performance and airflow. The air temperature split during typical conditions will be between 16 and 22 degrees' difference from the return to the supply. Keep

in mind that when you are doing a new system start-up, high humidity will cause your air temperature split to be on the low side. Refer to the air temperature split and comfort considerations sheets for further information.

For systems that are set to 400 CFM per ton, you can use a target delta T sheet like the one shown below.

| | Return Air Wet-Bulb (°F) (T return, wb) | | | | | | | | | | | | | | | F) (T , | eturn, wt |) | | | | | | | 647 | | | |
|--------|---|-------------|------|----------|------|------|------|------|------|------|------|------|------|------|------|---------|-----------|------|------|------|------|------|------|------|------|------|------|------|
| | | 50 | 51 | 52 | 53 | 54 | 55 | 56 | 57 | 58 | 59 | 60 | 61 | 62 | 63 | 64 | 65 | 66 | 67 | 68 | 69 | 70 | 71 | 72 | 73 | 74 | 75 | 76 |
| | 70 | 20.9 | 20.7 | 20.6 | 20.4 | 20.1 | 19.9 | 19.5 | 19.1 | 18.7 | 18.2 | 17.7 | 17.2 | 16.5 | 15.9 | 15.2 | 14.4 | 13.7 | 12.8 | 11.9 | 11.0 | 10.0 | 9.0 | 7.9 | 6.8 | 5.7 | 4.5 | 3.2 |
| | 71 | 21.4 | 21.3 | 21.1 | 20.9 | 20.7 | 20.4 | 20.1 | 19.7 | 19.3 | 18.8 | 18.3 | 17.7 | 17.1 | 16.4 | 15.7 | 15.0 | 14.2 | 13.4 | 12.5 | 11.5 | 10.6 | 9.5 | 8.5 | 7.4 | 6.2 | 5.0 | 3.8 |
| | 72 | 21.9 | 21.8 | 21.7 | 21.5 | 21.2 | 20.9 | 20.6 | 20.2 | 19.8 | 19.3 | 18.8 | 18.2 | 17.6 | 17.0 | 16.3 | 15.5 | 14.7 | 13.9 | 13.0 | 12.1 | 11.1 | 10.1 | 9.0 | 7.9 | 6.8 | 5.6 | 4.3 |
| (m) | 73 | 22.5 | 22.4 | 22.2 | 22.0 | 21.8 | 21.5 | 21.2 | 20.8 | 20.3 | 19.9 | 19.4 | 18.8 | 18.2 | 17.5 | 16.8 | 16.1 | 15.3 | 14.4 | 13.6 | 12.6 | 11.7 | 10.6 | 9.6 | 8.5 | 7.3 | 6.1 | 4.8 |
| return | 74 | 23.0 | 22.9 | 22.8 | 22.6 | 22.3 | 22.0 | 21.7 | 21.3 | 20.9 | 20.4 | 19.9 | 19.3 | 18.7 | 18.1 | 17.4 | 16.6 | 15.8 | 15.0 | 14.1 | 13.2 | 12.2 | 11.2 | 10.1 | 9.0 | 7.8 | 6.6 | 5.4 |
| E | 75 | 23.6 | 23.5 | 23.3 | 23.1 | 22.9 | 22.6 | 22.2 | 21.9 | 21.4 | 21.0 | 20.4 | 19.9 | 19.3 | 18.6 | 17.9 | 17.2 | 16.4 | 15.5 | 14.7 | 13.7 | 12.7 | 11.7 | 10.7 | 9.5 | 8.4 | 7.2 | 5.9 |
| °F (°F | 76 | 24.1 | 24.0 | 23.9 | 23.7 | 23.4 | 23.1 | 22.8 | 22.4 | 22.0 | 21.5 | 21.0 | 20.4 | 19.8 | 19.2 | 18.5 | 17.7 | 16.9 | 16.1 | 15.2 | 14.3 | 13.3 | 12.3 | 11.2 | 10.1 | 8.9 | 7.7 | 6.5 |
| Bull | 77 | 5 -2 | 24.6 | 24.4 | 24.2 | 24.0 | 23.7 | 23.3 | 22.9 | 22.5 | 22.0 | 21.5 | 21.0 | 20.4 | 19.7 | 19.0 | 18.3 | 17.5 | 16.6 | 15.7 | 14.8 | 13.8 | 12.8 | 11.7 | 10.6 | 9.5 | 8.3 | 7.0 |
| -fuc | 78 | 100 | 2 | 12 | 24.7 | 24.5 | 24.2 | 23.9 | 23.5 | 23.1 | 22.6 | 22.1 | 21.5 | 20.9 | 20.2 | 19.5 | 18.8 | 18.0 | 17.2 | 16.3 | 15.4 | 14.4 | 13.4 | 12.3 | 11.2 | 10.0 | 8.8 | 7.6 |
| Air | 79 | - | - | - | - | - | 24.8 | 24.4 | 24.0 | 23.6 | 23.1 | 22.6 | 22.1 | 21.4 | 20.8 | 20.1 | 19.3 | 18.5 | 17.7 | 16.8 | 15.9 | 14.9 | 13.9 | 12.8 | 11.7 | 10.6 | 9.4 | 8.1 |
| Ę | 80 | 150 | - | <u>.</u> | 100 | - | . a. | 25.0 | 24.6 | 24.2 | 23.7 | 23.2 | 22.6 | 22.0 | 21.3 | 20.6 | 19.9 | 19.1 | 18.3 | 17.4 | 16.4 | 15.5 | 14.4 | 13.4 | 12.3 | 11.1 | 9.9 | 8.7 |
| Ret | 81 | | - | | (**) | ्न | - | 1875 | 25.1 | 24.7 | 24.2 | 23.7 | 23.1 | 22.5 | 21.9 | 21.2 | 20.4 | 19.6 | 18.8 | 17.9 | 17.0 | 16.0 | 15.0 | 13.9 | 12.8 | 11.7 | 10.4 | 9.2 |
| | 82 | | - | | (H) | | - | | | 25.2 | 24.8 | 24.2 | 23.7 | 23.1 | 22.4 | 21.7 | 21.0 | 20.2 | 19.3 | 18.5 | 17.5 | 16.6 | 15.5 | 14.5 | 13.4 | 12.2 | 11.0 | 9.7 |
| | 83 | . | - | | | - | - | - | | - | 25.3 | 24.8 | 24.2 | 23.6 | 23.0 | 22.3 | 21.5 | 20.7 | 19.9 | 19.0 | 18.1 | 17.1 | 16.1 | 15.0 | 13.9 | 12.7 | 11.5 | 10.3 |
| | 84 | 940 1 | 2 | 3 | (2) | - | 2 | 343 | 1 | ×. | 25.9 | 25.3 | 24.8 | 24.2 | 23.5 | 22.8 | 22.1 | 21.3 | 20.4 | 19.5 | 18.6 | 17.6 | 16.6 | 15.6 | 14.4 | 13.3 | 12.1 | 10.8 |

If the leaving temperature/delta T split is high, it is an indication of low airflow. If it is low, it is an indication of poor system performance/capacity.

Again, this only applies to 400 CFM per ton. Systems set at 350 CFM per ton or less are more common today than ever, especially in humid climates. In those cases, the above chart won't apply, and the delta T will be higher.

Diagnosing With The Five Pillars

This list must be utilized by taking all five calculations and matching up the potential problems until you find the most likely ones. A very critical thing to remember is that a TXV system will maintain a constant superheat and fairly constant suction pressure. The exceptions to this rule are when the TXV fails; it is not receiving a full line of liquid, or it does not have the required liquid pressure/pressure drop to operate. This situation would show 0° subcooling, and in this case, it will no longer be able to maintain the correct superheat. Before using this list, you must also know what type of metering device is being utilized, then adjust your thinking accordingly. Also, remember that in heat mode, the condenser is inside while the evaporator is outside.

Low Suction Pressure

• Low on charge

• Low airflow/load; dirty filter, dirty evaporator, kinked return, return too small, not enough supply ducts, blower wheel dirty, blower not running correct speed, insulation pulling up against the blower, etc.

 Metering device restricting flow too much; piston too small, piston or TXV restricted, TXV failing closed

- Liquid line restriction; clogged filter/drier, clogged screen, kinked copper
- Low ambient (low evaporator load)
- Extremely kinked suction line (after the kink)
- Internal evaporator restriction

High Suction Pressure

- Overcharge
- High return temperature (evaporator load)

• Metering device allowing too much refrigerant flow; piston too large, TXV failing open, piston seating improperly

• Too much airflow over the evaporator (blower tapped or set too high)

• Compressor not pumping properly; leaking suction valve, leaking discharge valve, other compression issues

- Reversing valve bypassing
- Discharge line restriction

Low Head Pressure

- Low on charge
- · Low ambient temperature/low load
- Metering device allowing too much refrigerant flow; piston too large, TXV failing open,
- piston seating improperly
- Wet condenser coil

• Compressor not pumping properly; leaking suction valve, leaking discharge valve, other compression issues

- Reversing valve bypassing (heat pump units)
- Kinked suction line
- Restricted discharge line
- Severe liquid line restriction
- Wet condensing coil

High Head Pressure

Overcharge

• Low condenser airflow; condensing fan not operating, dirty condenser, fins bent on the condenser, bushes too close to the condenser, wrong blade, wrong motor, blade set wrong

- High outdoor ambient temperature
- Mixed/incorrect refrigerant or retrofit without proper markings
- Non-condensables in the system
- Liquid line restriction + overcharge (someone added charge when they saw low

suction) - piston too small, piston or TXV restricted, TXV failing closed, restricted line drier

Low Superheat

Overcharge

• Low airflow/load; dirty filter, dirty evaporator, kinked return, return too small, not enough supply ducts, blower wheel dirty, blower not running correct speed, insulation pulling up against the blower, etc.

• Metering device allowing too much refrigerant flow; piston too large, TXV failing open, piston seating improperly

- Low return air temperature
- Abnormally low humidity
- Internal evaporator restriction

• Very poor compression (compressor, reversing valve issues) but will also be combined with VERY HIGH suction

High Superheat

• Low on charge

 Metering device restricting flow/underfeeding/over-metering; piston too small, piston or TXV restricted, TXV failing closed

- High return air temperature
- Liquid line restriction; clogged filter/drier, clogged screen, kinked copper

Low Subcooling

• Low on charge

• Metering device allowing too much refrigerant flow; piston too large, TXV failing open, piston seating improperly

• Compressor not pumping properly; leaking suction valve, leaking discharge valve, bad or broken crank

- Reversing valve bypassing
- Discharge line restriction
- Compressor not pumping

High Subcooling

Overcharge

• Metering device restricting too much flow – piston too small, piston or TXV restricted, TXV failing closed

• Liquid line restriction; clogged filter/drier, clogged screen, kinked copper

• Dirty condenser coil on new high-efficiency condensers (increased condensing temp can actually result in higher subcooling)

- Having an H.R.U. in the discharge line (old-school, I know)
- Internal evaporator restriction

High Evaporator Air Temperature Split

• Low airflow; dirty filter, dirty evaporator, kinked return, return too small, not enough supply ducts, blower wheel dirty, blower not running correct speed, insulation pulling up against the

blower, etc.

- Abnormally low humidity (WB Temp)
- Blower not running the correct speed or running backward

Low Evaporator Air Temperature Split

- Undercharge
- Severe overcharge with fixed orifice metering device; saturation temperature is increased with overcharge
- Metering device not functioning properly; restricting too much flow or allowing too much flow
- Too much airflow through the evaporator; blower not running correct speed
- Heat strips running with the air
- Abnormally high humidity
- Liquid line restriction
- Compressor not pumping properly; bad suction valve, bad discharge valve, bad or broken crank
- Reversing valve bypassing
- Discharge line restriction

This is an incomplete list designed to help you. Always keep your eyes and ears open for other possibilities. Diagnosis is an art as well as a science.

The <u>MeasureQuick</u> app is a great free app that can help you make a complete diagnosis using these five pillars and more.

—Bryan